



## **Specifying Unbalance and the Location of Tolerance Planes**



**Gary K. Grim**  
**Jake Schlaegel**

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**PRECISION MEASUREMENT AND TESTING EQUIPMENT AND SERVICES**

Balance Technology Inc • 7035 Jomar Drive, Whitmore Lake, MI 48189 • 734-769-2100 • USA  
[www.balancetechnology.com](http://www.balancetechnology.com)



## ***About Us***

Headquartered near Ann Arbor, Michigan (USA), Balance Technology Inc **BTI** is a thriving Precision Measurement and Testing company with a strong domestic and international presence. Since 1968, **BTI** has set the standard in industrial Precision Measurement & Testing systems. With 13,000 plus systems shipped worldwide, our team approach to customer satisfaction and technical innovations has forged our reputation as an industry leader and trusted partner.



**BTI World Headquarters.**

**All our equipment is engineered & manufactured in the USA.**

**BTI** engineers and manufactures a complete line of industrial precision measurement and testing equipment, including static and dynamic balancing equipment, dimensional gages, mass centering equipment, eddy current crack detection systems, surface finish measurement equipment, NVH equipment (noise vibration and harshness), functional test stands, spinners, motor testers, and resonant frequency measurement systems. We also engineer and manufacture specialized test systems, including torque-to-turn, backlash, end play, and destructive test equipment.

Additionally, **BTI's** unique ability to combine the aforementioned technologies into one fully integrated system enables our clients to reduce capital expenditures, increase product quality, and minimize floor space requirements.

Let our staff of over 50 engineers design a custom solution for your specific requirements. Furthermore, our commercial Measurement & Testing Services Group (M & T Services) can assist with everything from prototype testing, R&D work, master certification to running small to medium production runs.

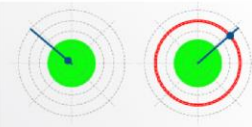
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## ***What we do...***

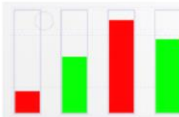
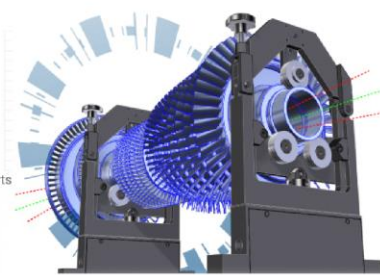
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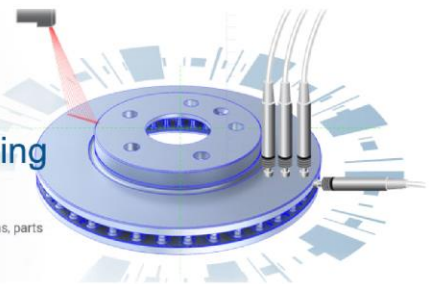
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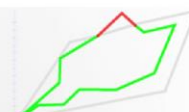
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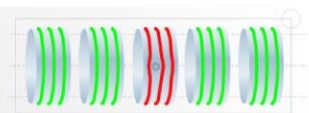
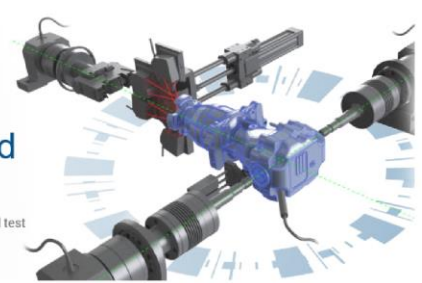
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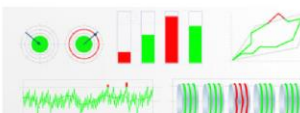
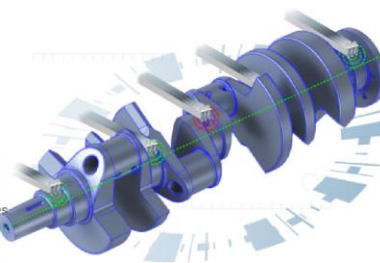
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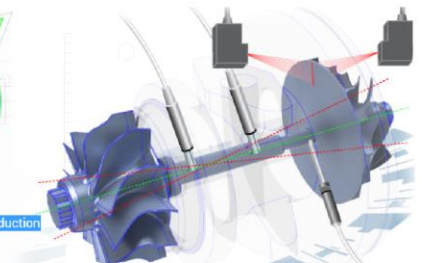
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## **SPECIFYING UNBALANCE AND THE LOCATION OF TOLERANCE PLANES**

Gary K. Grim, Bruce J. Mitchell, Jr.

Unbalance can be specified in many forms. The most common is expressed as a weight of material to be added or removed at a specified correction radius. The weight units can be any convenient units; grams (g), ounces (oz), and kilograms (kg) are common units. Occasionally Newton's (N) are specified, but for practical use must be converted to available weight scale units. Length units are often expressed in; inches (in), millimeters (mm), centimeters (cm), and meters (m). The most common combinations used to specify unbalance are ounce-inches (oz-in), gram-inches (g-in), gram-millimeters (g-mm), gram-centimeters (g-cm), and kilogram-meters (kg-m).

A complete specification for two-plane unbalance must include both the maximum allowable unbalance and the location of the tolerance planes. If the locations are omitted, the specification is incomplete and the locations must be assumed. If the workpiece is to be balanced dynamically, a correction method and correction location should be identified as well.

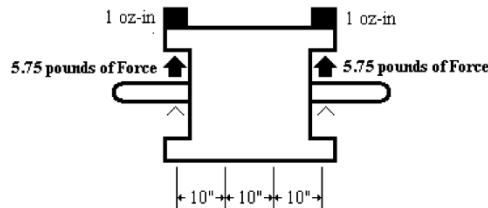
In the past, balancers could only operate in one set of planes, so it was only possible to audit unbalance at the correction planes, therefore requiring the location of the tolerance planes to coincide with the correction planes. However, modern balancers incorporating PC's allow the tolerance planes to be different than the correction planes.

So where should the unbalance be audited? At the bearing planes! After all, the forces due to unbalance will be "felt" through the bearings. When the balance tolerance is established at the bearing planes, the force of unbalance is the same whether it be static or couple unbalance. Whereas, if the balance tolerance is established at planes narrower than the bearing planes, the force on the bearings due to couple is less than the force due to static unbalance. This is caused from the couple component being balanced unnecessary low as compared to the force unbalance. One additional thing to consider when correcting for couple unbalance; when the correction plans are as far apart as possible, a minimum amount of correction required.

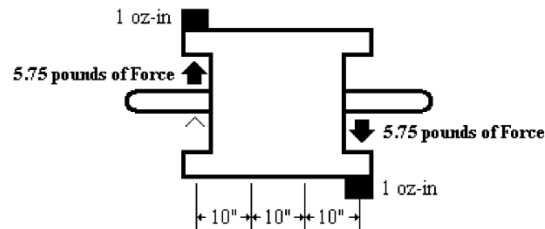
Consider the following examples:



A rotor spinning at 1800 rpm with 1.0 ounce-inch of unbalance directly in-line with the bearing planes will produce 5.75 pounds of centripetal force on each bearing regardless of the angle around the workpiece or the relative angle. The following two examples show the extremes, the first is with two weights at the same angle, the second is with the two weights 180 degrees opposed.

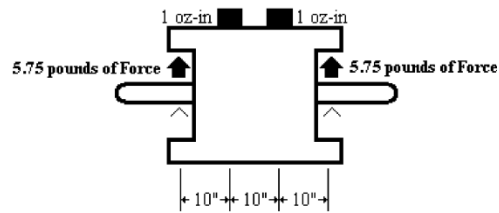


**Force** (single plane) unbalance is **2 oz in**  
**Left plane** unbalance at the left bearing plane **1 oz in**  
**Right plane** unbalance at the right bearing plane **1 oz in**  
**Couple** unbalance at the bearings **0 oz in<sup>2</sup>**.

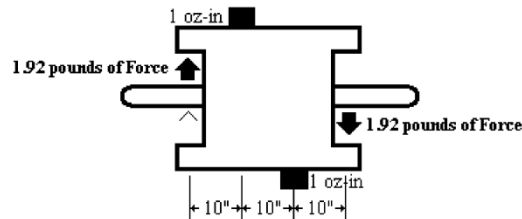


**Force** (single plane) unbalance is **0 oz in**  
**Left plane** unbalance at the left bearing plane **1 oz in**  
**Right plane** unbalance at the right bearing plane **1 oz in**  
**Couple** unbalance at the bearings **30 oz in<sup>2</sup>**.

This is not true when the unbalances are moved to the inner set of planes. Unbalances at the same angle produce the same static unbalance whereas the unbalance placed at opposite angles on the workpiece produce a reduced amount of pure couple. The effect of the couple at the bearings produces 1/3 of the force, because the distance between weight planes is 1/3 the distance between the bearing planes. Instead of 5.75 pounds, the force on the bearings is now 1.92 pounds.



**Force** (single plane) unbalance is **2 oz in**  
**Left plane** unbalance at the left bearing plane **1 oz in**  
**Right plane** unbalance at the right bearing plane **1 oz in**  
**Couple** unbalance at the bearings **0 oz in<sup>2</sup>**.



**Force** (single plane) unbalance is **0 oz in**  
**Left plane** unbalance at the left bearing plane **.33 oz in**  
**Right plane** unbalance at the right bearing plane **.33 oz in**  
**Couple** unbalance at the bearings **10 oz in<sup>2</sup>**.

Equal amounts of unbalance in these planes produce a force on the bearings which varies between 5.75# and 1.92# depending on the relative angle of the unbalances. If the balance tolerance is specified at planes 10 inches apart the maximum force on the bearings is 5.75# when the unbalances are aligned. Then at all other relative angles the tolerance is too low and the balance is being reduced lower than necessary.